## Solar 3D printer for Moon Base Preliminary Design Review

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#### Initial System Requirements

Telescope FOV: at least 32 arcminutes

Primary Mirror Diameter: 2 meters

Secondary Mirror Diameter: 0.5 meters

Mirror Reflectance: Greater than 80% from 250nm to 1000nm

Telescope F/#: 2

Mirror Materials: Zerodur or ULE

Primary Mirror Mount: Must be able to achieve 50 nm rms from surface irregularities, self-weight deflection and mound induced deflection for zenith to horizon pointing

Mirror Mount and Secondary mirror support Material: Low CTE Graphite Epoxy

Operational Position Stability: less than 1 arminute tip/tilt, and 50 µm decenter

Angle of incidence at Fiber Optic interface: 14.5 degrees

Beam Width at Fiber Optic interface: less than 350 µm

Focal plane spot size at printing surface: 50  $\mu$ m

#### System Survival Requirements

Operational Environment

Temperature: -100C to 100C

Gravity: 1/6 Earth gravity

Survival:

Shock: 40G

Temperature: -100C to 100C

Limitations

Weight limitations: Less than 50 kg

# System Changes for Feasibility and simplification

Primary Mirror: 1 meter diameter

Secondary Mirror: removed

Mirror Reflectance: 400-1000 nm

RMS error: <100 nm

Weight Limitation: 50 kg for mirror, 100 kg for mount

Effective light collection area: 0.785 meters

With reduction in wavelength and surface area the final throughput of the system can no longer exceed 1 kW.

This may pose a problem with melting the substrate, but can be compensated for by slowing down the feedrate for the 3D printer positioning system

#### Design Concept



Primary mirror gathers sunlight and passes the light through a fiber optic cable to the 3D printer area, where the light is focused into a beam 50 microns wide



#### Design Choices – Mirror Mount

Vertical mount preference: Six point contact

 Self weight deflection calculations for a six-point mount when pointed to zenith

• 
$$\delta_{V max} = C_{SP} * \left(\frac{\rho g}{E}\right) * \frac{r^4}{h^2} * (1 - \nu^2)$$

- To achieve necessary rms limitation 2 meter mirror weight too high
- 1 meter mirror will have rms≈30nm



#### Design Choices – Mirror Mount

Horizontal Mount choice: Whiffle-tree mount

Self-weight calculations for whiffle tree mount when pointed to horizon

• 
$$\delta_{H\,rms} = (a_0 + a_1\gamma + a_2\gamma^2) * \left(\frac{2\rho g}{E}\right) * r^2$$

- Required RMS could not be achieved with 2 meter mirror
- I meter mirror will have rms≈33 nm

Total rms from self weight deflection will be:  $\delta_{rms} = \sqrt{(\delta_{V rms} \cos\theta)^2 + \delta_{H rms} \sin\theta)^2}$ 



## Design Choices – Primary Mirror

Primary mirror radius of curvature of 4 m, with a conic constant of -1.005

- Eliminates the need for a secondary mirror
- Meets the requirements for fiber acceptance angle of 14.5 degree (angle at focal point is 14 degrees)
- Allows for looser tolerances on Primary mirror pointing and stability

#### Primary mirror with silver coating

- Helps mitigate chromatic aberrations
- Higher reflectance than aluminum from 400 nm
- Loss of reflectance below 400 nm

#### Design Choices – Mount Materials

For the mirror mount, a standard uni-direction carbon fiber reinforced plastic will be used.

- Low CTE of 0.57
  - For temperature variation of ±100 Celcius, ΔL will result in 0.1 mm change in lateral fiber optic location
- Strength and rigidity
  - Change in fiber location of less than 50 microns at horizon
  - Change in fiber location of less than 2 microns at zenith

#### Design Choices – Adhesive Bonds

- Use RTV112 for binding rockers to the mirror
- Thickness of 100 microns
- Shear stress due to thermal expansion/contraction

• 
$$au = rac{Ga}{2t} (lpha_1 - lpha_2) \Delta T = 0.87$$
 kPa per bond

Stress from 40 G shock

•  $\sigma = \frac{m*40g}{A} = 0.82$  MPa per bond (below the tensile strength of 2.24 MPa, and within the safety factor)

#### Design Choices – Printer Lenses

Use achromatic doublets to reduce chromatic aberations

- Use BK7 and SF5 due to their ability to transmit light from 400-1000 nm
- Lenses to be supported in a barrel, placed vertically
- Can make small adjustments to final lens for focal plane fine adjustments
- Barrel made from copper due to high thermal conductivity
  - Should help mitigate heat build up in barrel

Barrel will be located in an enclosed area, so not subject to outside temperature variations



## Design Choice – Lens Tolerances

Required tolerances for a 50 micron spot from Zemax

- Lens element spacings less than 0.05 mm
- Lens element decenters less than 0.1 mm
- Lens element tilts less than 0.4 mm
- Individual Lens specifications
  - As long as each individual lens is made to high precision optical quality standards, then tolerances will not be an issue

## Design Choices – Fiber Optic Cable

#### Fiber cable specifications

- Needs little to no attenuation from 400-1000 nm for a distance of 500m
- Pure fused silica core/fluorine doped silica cladding will provide optimal performance
- Gold coating will perform under required temperature variances (works from -263 to 700 C)
- Fiber with gold coatings can be procured at diameter of up to 400 microns
- Fibers with acceptance angles up to 30 degrees can be procured

#### Design Choices – Fiber Optic Cable



#### Preliminary Mount and Mirror Design



#### Preliminary Mount and Mirror Design



## Additional Calculations

- What Finite Element Analysis has shown
- Deflection of fiber location on telescope mount
  - 1.95 microns at zenith
  - 50 microns at horizon

#### Hand Calculations

- Deflection of fiber location on due to temperature variance
  - $\Delta L = \alpha L \Delta T = 108$  microns (Along the optical axis)
- System Throughput:
  - 93% of light between 400 and 1000 nm will get through
  - Total throughput of ~700kW

#### Remaining Work

Perform Finite Element Analysis of all models

Finalize Lens design (able to achieve 60 micron spot size with wavelength from 400-2000nm so far)

- Finalize Lens Barrel Design
- Calculate throughput of the system
- Finalize mount design
- Finalize Fiber interfaces, specifically material
- Define manufacture and assembly procedure